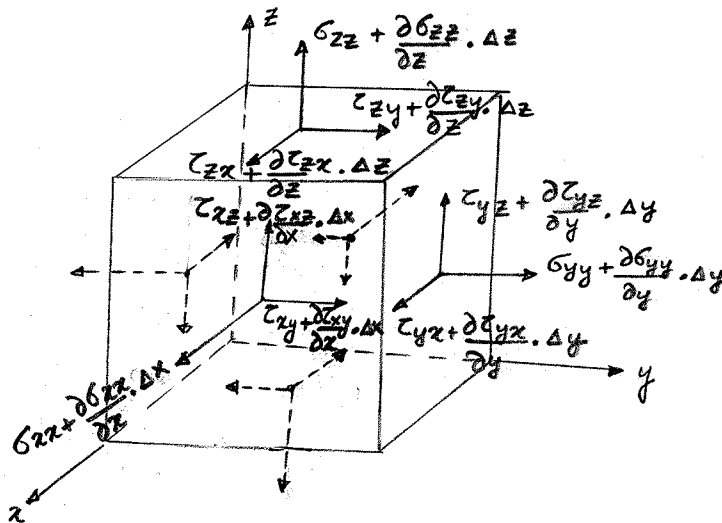


• Stress Tensor

$$\begin{bmatrix} \sigma_{xx} & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_{yy} & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_{zz} \end{bmatrix}$$



Stress block at point Q

• Equations of Equilibrium:

$$\begin{aligned} \frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} + \frac{\partial \tau_{xz}}{\partial z} + \bar{F}_x &= 0 \\ \frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_{yy}}{\partial y} + \frac{\partial \tau_{yz}}{\partial z} + \bar{F}_y &= 0 \\ \frac{\partial \tau_{xz}}{\partial x} + \frac{\partial \tau_{yz}}{\partial y} + \frac{\partial \sigma_{zz}}{\partial z} + \bar{F}_z &= 0 \end{aligned}$$

} 3 at each point in the body.

• Strain - displacement relations:

$$\begin{aligned} \epsilon_{xx} &= \frac{\partial u}{\partial x} & \gamma_{xy} &= \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \\ \epsilon_{yy} &= \frac{\partial v}{\partial y} & \gamma_{yz} &= \frac{\partial v}{\partial z} + \frac{\partial w}{\partial y} \\ \epsilon_{zz} &= \frac{\partial w}{\partial z} & \gamma_{xz} &= \frac{\partial u}{\partial z} + \frac{\partial w}{\partial x} \end{aligned}$$

} $\epsilon \rightarrow$ normal strains
 $\gamma \rightarrow$ shear strains

} $\left. \begin{matrix} u(x, y, z) \\ v(x, y, z) \\ w(x, y, z) \end{matrix} \right\}$ displacement

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \tau_{xy} \\ \tau_{yz} \\ \tau_{xz} \end{Bmatrix} = \begin{bmatrix} 1-\nu & \nu & \nu & 0 & 0 & 0 \\ \nu & 1-\nu & \nu & 0 & 0 & 0 \\ \nu & \nu & 1-\nu & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{2}(1-2\nu) & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{2}(1-2\nu) & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{2}(1-2\nu) \end{bmatrix} \begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{zz} \\ \gamma_{xy} \\ \gamma_{yz} \\ \gamma_{xz} \end{Bmatrix} \times \frac{E}{(1-2\nu)(1+\nu)}$$

$$\{\sigma\}_{6 \times 1} = [C]_{6 \times 6} \{\epsilon\}_{6 \times 1}$$

STRESS-STRAIN RELATIONS

• PLANE STRESS:

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{Bmatrix} = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1}{2}(1-\nu) \end{bmatrix} \begin{Bmatrix} \epsilon_x \\ \epsilon_y \\ \gamma_{xy} \end{Bmatrix}$$

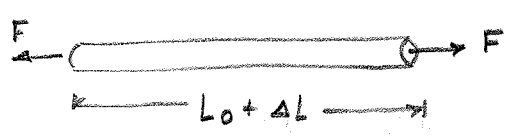
$$\epsilon_z = -\frac{\nu}{1-\nu} (\epsilon_x + \epsilon_y) \quad \text{and} \quad \sigma_z = 0$$

• PLANE STRAIN:

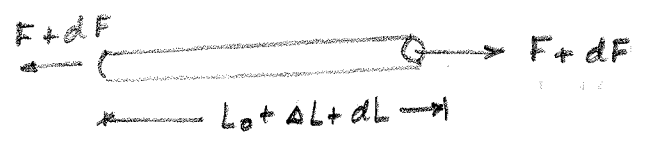
$$\begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \gamma_{xy} \end{Bmatrix} = \frac{1-\nu^2}{E} \begin{bmatrix} 1 & -\frac{\nu}{1-\nu} & 0 \\ -\frac{\nu}{1-\nu} & 1 & 0 \\ 0 & 0 & \frac{2}{1-\nu} \end{bmatrix} \begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{Bmatrix}$$

$$\epsilon_z = 0 \quad \text{and} \quad \sigma_{zz} = \nu(\sigma_{xx} + \sigma_{yy})$$

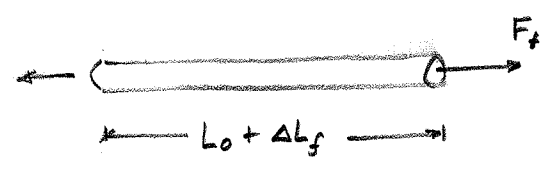
Strain Energy:



$$\epsilon = \frac{\Delta L}{L_0} ; \sigma = \frac{F}{A_0} ; W_1 = F \cdot \Delta L$$



$$\epsilon = \frac{\Delta L + dL}{L_0} ; \sigma = \frac{F + dF}{A_0} ; dW = (F + dF) \cdot dL$$



$$\epsilon_f = \frac{\Delta L_f}{L_0} ; \sigma = \frac{F_f}{A_0} ; W_f = F_f \cdot \Delta L_f$$

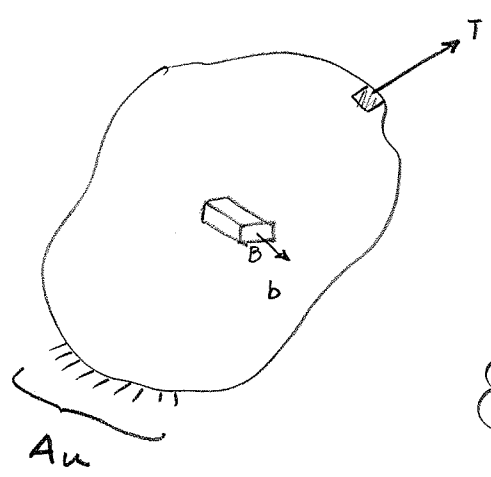
$$dW = (F + dF) \cdot dL = F dL$$

$$\therefore W = \int_0^{\epsilon_f} F dL = \int_0^{\epsilon_f} \sigma \cdot A_0 \cdot d\epsilon \cdot L_0 = (A_0 L_0) \int_0^{\epsilon_f} \sigma \cdot d\epsilon$$

$$\bar{u} = u/v_0 = \int_{\epsilon_0}^{\epsilon_f} \sigma_x d\epsilon_x$$

$$u = \int_{vol} \bar{u} dV$$

$$\bar{u} = \int_{\epsilon_0}^{\epsilon_f} \sigma_x d\epsilon_x + \int_{\epsilon_0}^{\epsilon_f} \sigma_y d\epsilon_y + \int_{\epsilon_0}^{\epsilon_f} \tau_{xy} d\gamma_{xy} + \int_{\epsilon_0}^{\epsilon_f} \tau_{yz} d\gamma_{yz} + \int_{\epsilon_0}^{\epsilon_f} \tau_{xz} d\gamma_{xz}$$



Body force $[b]$
 Surface traction $\{T\}$
 virtual displacement $\{\delta u\}$

$\delta W_e = \delta W_i$

 Equilibrium equation

$$\delta W_e = \int_{Vol} [b] \{\delta u\} dV + \int_{Surf} \{T\} \{\delta u\} dA$$

$$\delta W_e = \int_{Vol} [b] \{\delta u\} dV + \int_{Surf} [\sigma] \{\hat{n}\} \{\delta u\} dA \quad \text{--- (1)}$$

$$\delta W_i = \delta U = \int_{Vol} \overset{\text{virtual disp.}}{\bar{u}} dV$$

$$= \int_{Vol} (\sigma_x \cdot \delta \epsilon_x + \sigma_y \cdot \delta \epsilon_y + \sigma_z \cdot \delta \epsilon_z + \tau_{xy} \cdot \delta \gamma_{xy} + \tau_{yz} \cdot \delta \gamma_{yz} + \tau_{xz} \cdot \delta \gamma_{xz})$$

$$\int_{Vol} f(x, y, z) \cdot \frac{\partial g(x, y, z)}{\partial x} dV = \int_{Surf} f(x, y, z) \cdot g(x, y, z) n_x dA - \int_{Vol} g(x, y, z) \cdot \frac{\partial f}{\partial x}(x, y, z) dV$$

Consider: $\int_{Vol} \sigma_x \cdot \delta \epsilon_x$ $\delta \epsilon_x = \frac{\partial(\delta u)}{\partial x}$

$$= \int_{Surf} (\sigma_x \cdot \delta u) n_x dA - \int_{Vol} \delta u \cdot \left(\frac{\partial \sigma_x}{\partial x}\right) \cdot dV$$

Consider: $\int_{Vol} (\tau_{xy} \cdot \delta \gamma_{xy}) dV$ $\delta \gamma_{xy} = \frac{\partial(\delta v)}{\partial x} + \frac{\partial(\delta u)}{\partial y}$

$$= \int_{Vol} \tau_{xy} \cdot \frac{\partial(\delta v)}{\partial x} dV + \int_{Vol} \tau_{xy} \cdot \frac{\partial(\delta u)}{\partial y} dV$$

$$= \int_{Surf} \tau_{xy} \cdot \delta v \cdot n_x dA - \int_{Vol} \delta v \cdot \frac{\partial \tau_{xy}}{\partial x} \cdot dV + \int_{Surf} (\tau_{xy} \cdot \delta u \cdot n_y) dA - \int_{Vol} \delta u \cdot \frac{\partial \tau_{xy}}{\partial y} \cdot dV$$

collecting terms.

$$\delta W_i = \int_{\text{surf.}} [\sigma] \{\hat{n}\} \{\delta u\} dA - \int_{\text{vol.}} \begin{bmatrix} \frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} + \frac{\partial \tau_{xz}}{\partial z} \\ - \\ - \end{bmatrix} \cdot \{\delta u\} dV$$

$$\delta W_i = \delta W_e.$$

$$\Rightarrow \int_{\text{surf.}} [\sigma] \{\hat{n}\} \cdot \{\delta u\} dA - \int_{\text{vol.}} [A] \{\delta u\} dV = \int_{\text{vol.}} [b] \cdot \{\delta u\} dV + \int_{\text{surf.}} [\sigma] \{\hat{n}\} \cdot \{\delta u\} dA$$

$$\therefore \int_{\text{vol.}} \{[A] + [b]\} \{\delta u\} dV = 0$$

For arbitrary $\{\delta u\}$

$$[A] + [b] = 0$$

→ equilibrium equations.

Weak form of static equilibrium

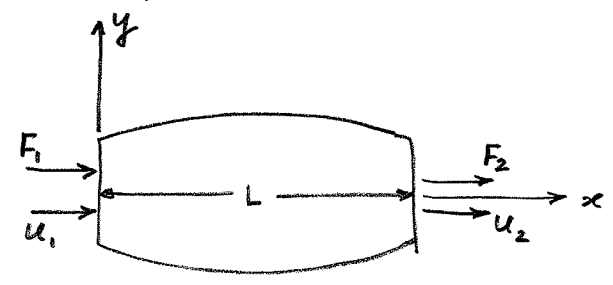
Example: Rayleigh-Ritz Method: a specific way of implementing the Principle of Virtual Displacements

Total Potential Energy = $\Pi = W_i - W_e$

Principle of virtual displacements $\delta\Pi = \delta W_i - \delta W_e = 0$

Approximate the displacements as a sum of known functions with unknown coefficients

Use the virtual displacement as the same as real displacements.



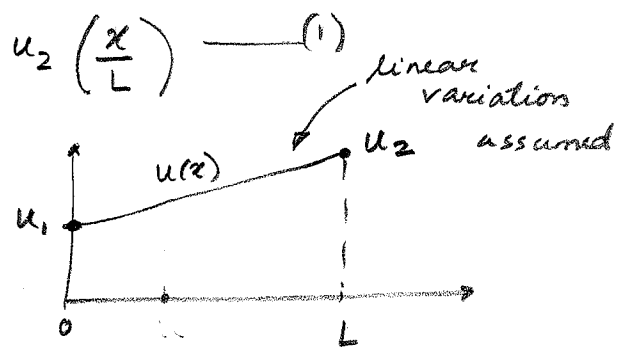
Given: Axial bar of length 'L', constant modulus of elasticity E, $A(x) = A_0 \times (1 + \beta \sin \frac{\pi x}{L})$ β is known

F_1 and F_2 act @ $x=0$ & $x=L$

corresponding deflections are u_1 & u_2

Solve using Rayleigh-Ritz

Assume $u(x) = u_1 (1 - \frac{x}{L}) + u_2 (\frac{x}{L})$ — (1)



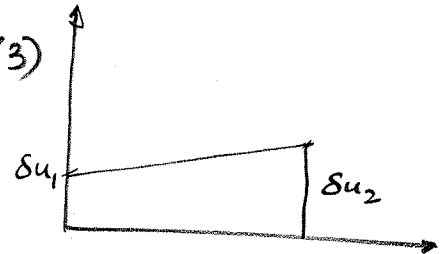
$$\therefore \epsilon_x = \frac{\partial u}{\partial x} = u_1 \left(-\frac{1}{L}\right) + \frac{u_2}{L} = \frac{u_2 - u_1}{L}$$

$$\therefore \sigma_x = E \times \left(\frac{u_2 - u_1}{L}\right) \quad \text{————— (2)}$$

To this system in equilibrium, apply a virtual displacement

$$\delta u(x) = \delta u_1 \left(1 - \frac{x}{L}\right) + \delta u_2 \left(\frac{x}{L}\right) \quad \text{— (3)}$$

$$\therefore \delta \epsilon_x = \frac{\partial (\delta u)}{\partial x} = -\frac{\delta u_1}{L} + \frac{\delta u_2}{L}$$



$$\delta \epsilon_x = \frac{\delta u_2 - \delta u_1}{L} \quad \text{— (4) virtual strain due to virtual disps.}$$

$$\delta W_e = F_1 \times \delta u_1 + F_2 \delta u_2 \quad \text{————— (5)}$$

$$\delta W_i = \int \sigma_x \cdot \delta \epsilon_x dV = \int_0^L E \times \left(\frac{u_2 - u_1}{L}\right) \times \left(\frac{\delta u_2 - \delta u_1}{L}\right) \times A(x) dx$$

$$= E \left(\frac{u_2 - u_1}{L}\right) \left(\frac{\delta u_2 - \delta u_1}{L}\right) \times \int_0^L A_0 \left(1 + \beta \sin \frac{\pi x}{L}\right) \cdot dx$$

$$= A_0 E \left(\frac{u_2 - u_1}{L}\right) \left(\frac{\delta u_2 - \delta u_1}{L}\right) \times \left[x + \beta \times \frac{(-\cos \pi x / L)}{\pi / L} \right]_0^L$$

$$= A_0 E \left(\frac{u_2 - u_1}{L}\right) \left(\frac{\delta u_2 - \delta u_1}{L}\right) \times \left[L + \beta \frac{1}{\pi / L} - \left(0 + \beta \times \frac{-1}{\pi / L}\right) \right]$$

$$= A_0 E \left(\frac{u_2 - u_1}{L}\right) \left(\frac{\delta u_2 - \delta u_1}{L}\right) \times \left(L + \frac{2\beta L}{\pi} \right)$$

$$\therefore \delta W_i = A_0 E \frac{(u_2 - u_1)}{L} \times \left(1 + \frac{2\beta}{\pi}\right) (\delta u_2 - \delta u_1)$$

$$\delta W_i = -A_0 E \frac{(u_2 - u_1)}{L} \left(1 + \frac{2\beta}{\pi}\right) \delta u_1 + A_0 E \frac{(u_2 - u_1)}{L} \left(1 + \frac{2\beta}{\pi}\right) \delta u_2$$

$$\delta W_e = \delta W_i$$

$$\therefore F_1 \delta u_1 + F_2 \delta u_2 = \text{---} \delta u_1 + \text{---} \delta u_2$$

$$\begin{aligned} F_1 &= -A_0 E \left(\frac{u_2 - u_1}{L}\right) \left(1 + \frac{2\beta}{\pi}\right) \\ F_2 &= EA_0 \left(\frac{u_2 - u_1}{L}\right) \left(1 + \frac{2\beta}{\pi}\right) \end{aligned}$$

Since δu_1 & δu_2 are arbitrary

$$\begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix} = \frac{EA_0}{L} \left(1 + \frac{2\beta}{\pi}\right) \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \end{Bmatrix}$$

Solution

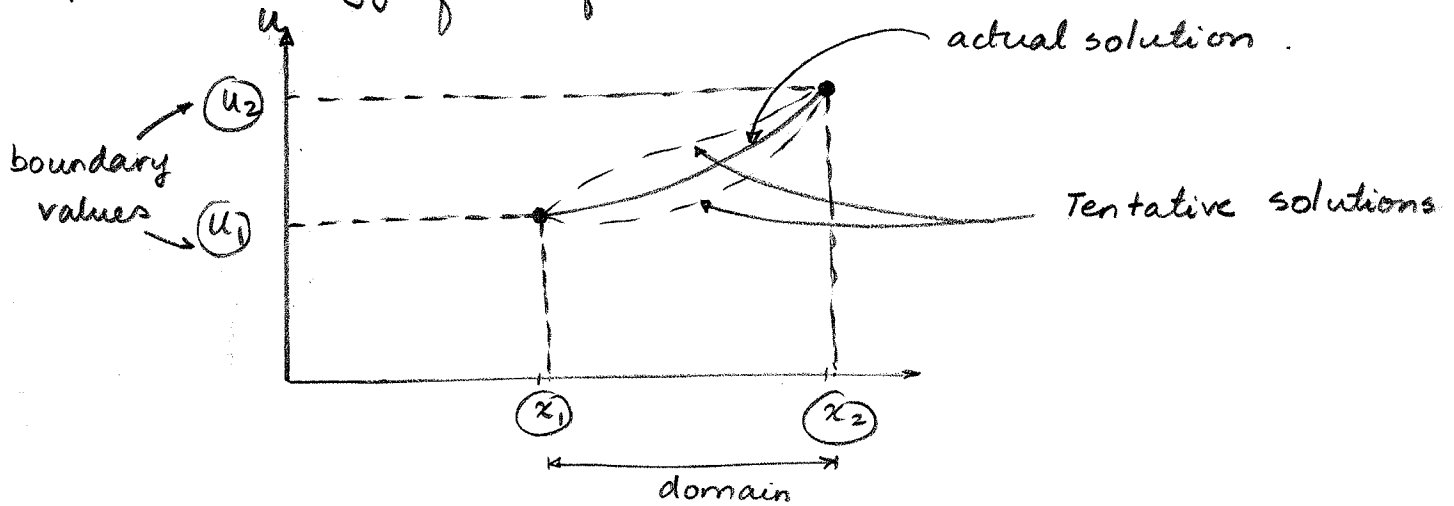
VARIATIONAL METHODS

(1) FUNCTIONAL $A = \int_{x_1}^{x_2} F(x, u, u', u'') dx$ (1)

function of other functions

$u \rightarrow u(x)$
 $u' \rightarrow \frac{d}{dx} u(x)$
 $u'' \rightarrow \frac{d^2}{dx^2} u(x)$

In mechanics, the functional has a clear meaning, e.g., the potential energy of a deformable solid.



The integral (1) is defined in the region or domain $[x_1, x_2]$

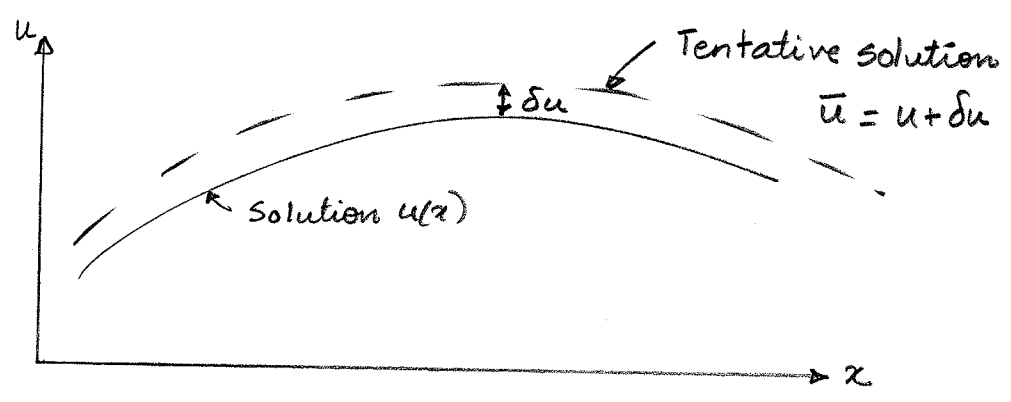
$u(x_1) = u_1$ & $u(x_2) = u_2$ ← boundary values.

- The actual solution for $u(x)$ is not known!
- A tentative solution is tried for the problem, and the functional is expressed in terms of tentative solution.
- From all such possible solutions that satisfy the boundary conditions, the solution which satisfies the variational principle governing the behavior will be the one that makes the functional A stationary.

- The mathematical procedure used to select the correct solution from a number of tentative solutions is called the calculus of variations.
- Any tentative solution \bar{u} in the neighborhood of the exact solution, may be represented as the sum of the exact solution u and a variation of u , δu

$$\bar{u} = u + \delta u$$

- The variation in $u = u(x)$ is defined as an infinitesimal change in u for a fixed value of x , i.e., $\delta x = 0$
- Notation $\delta u \rightarrow$ variational notation.



- The δ can be treated as an operator, similar to the differential operator, d

- The variation operator is commutative with both differentiation and integration

$$\delta\left(\int F dx\right) = \int (\delta F) dx$$

variation of integral = integral of variation

$$\delta\left(\frac{du}{dx}\right) = d\left(\frac{\delta u}{dx}\right)$$

variation of derivative = derivative of variation

- Variation of a function of several variables or functional

$$\delta F = \frac{\partial F}{\partial u} \cdot \delta u + \frac{\partial F}{\partial u'} \cdot \delta u' + \frac{\partial F}{\partial u''} \cdot \delta u''$$

- Consider the functional $A = \int_{x_1}^{x_2} F dx$

Small variations in this functional A correspond to small variations in the solution.

By analogy to the max or min of simple functions in ordinary calculus, the vanishing of the first variation of the functional A will give the condition for A to be stationary.

$$\delta A = \int_{x_1}^{x_2} \delta F dx = \int_{x_1}^{x_2} \left(\frac{\partial F}{\partial u} \cdot \delta u + \frac{\partial F}{\partial u'} \cdot \delta u' + \frac{\partial F}{\partial u''} \cdot \delta u'' \right) dx = 0$$

Integration by parts for the respective terms:

$$\int_{x_1}^{x_2} \frac{\partial F}{\partial u} \cdot \delta u \, dx \quad \text{--- (a)}$$

$$\int_{x_1}^{x_2} \frac{\partial F}{\partial u'} \cdot \delta u' \, dx = \left[\frac{\partial F}{\partial u'} \cdot \delta u \right]_{x_1}^{x_2} - \int_{x_1}^{x_2} \frac{d}{dx} \left(\frac{\partial F}{\partial u'} \right) \delta u \, dx$$

and

$$\int_{x_1}^{x_2} \frac{\partial F}{\partial u''} \cdot \delta u'' \, dx = \left[\frac{\partial F}{\partial u''} \cdot \delta u' \right]_{x_1}^{x_2} - \left[\frac{d}{dx} \left(\frac{\partial F}{\partial u''} \right) \delta u \right]_{x_1}^{x_2} + \int_{x_1}^{x_2} \frac{d^2}{dx^2} \left(\frac{\partial F}{\partial u''} \right) \delta u \, dx$$

$$\begin{aligned} \delta A &= \int_{x_1}^{x_2} \left[\frac{\partial F}{\partial u} - \frac{d}{dx} \left(\frac{\partial F}{\partial u'} \right) + \frac{d^2}{dx^2} \left(\frac{\partial F}{\partial u''} \right) \right] \delta u \, dx \\ &+ \left[\frac{\partial F}{\partial u'} - \frac{d}{dx} \left(\frac{\partial F}{\partial u''} \right) \right] \delta u \Big|_{x_1}^{x_2} \\ &+ \left[\left(\frac{\partial F}{\partial u''} \right) \delta u' \right]_{x_1}^{x_2} = 0 \end{aligned}$$

The variation δu is arbitrary;

$$\therefore \frac{\partial F}{\partial u} - \frac{d}{dx} \left(\frac{\partial F}{\partial u'} \right) + \frac{d^2}{dx^2} \left(\frac{\partial F}{\partial u''} \right) = 0 \quad \text{--- (1)}$$

$$\left[\frac{\partial F}{\partial u'} - \frac{d}{dx} \left(\frac{\partial F}{\partial u''} \right) \right] \delta u \Big|_{x_1}^{x_2} = 0 \quad \text{--- (2)}$$

$$\left[\left(\frac{\partial F}{\partial u''} \right) \delta u' \right]_{x_1}^{x_2} = 0 \quad \text{--- (3)}$$

Equation (1) is the governing differential equation and is called the EULER equation or the Euler-Lagrange equation.

$$\left[\frac{\partial F}{\partial u'} - \frac{d}{dx} \left(\frac{\partial F}{\partial u''} \right) \right]_{x_1}^{x_2} = 0$$

or $\delta u = 0$

forced boundary conditions

natural boundary conditions.

$$\text{and } \frac{\partial F}{\partial u''} = 0$$

or $\delta u' = 0$

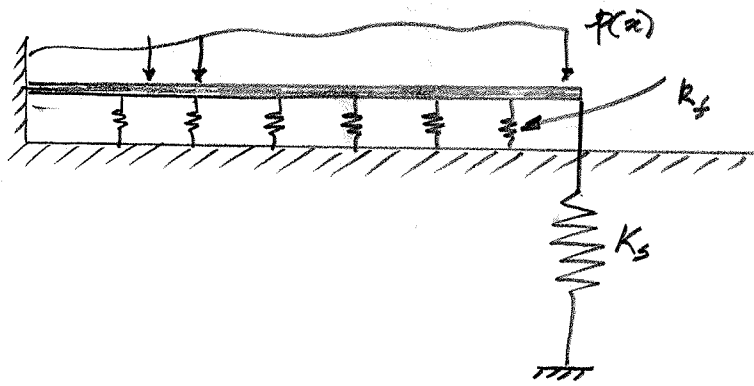
natural boundary conditions.

Either the natural or the free boundary conditions need to be satisfied at each end.

- CALCULUS OF VARIATIONS has produced one governing differential equation and free boundary conditions.
- When applied to the integral of a functional, the COV approach provides a complete description of the problem including the governing differential equation and inherent boundary conditions.

EULER-LAGRANGE FORMULATION:

Example: Beam on Elastic Foundation.



Beam with one end rigidly supported and the other end propped on an elastic spring

Geometric or forced boundary condition

$$\begin{cases} u(x=0) = 0 \\ u'(x=0) = 0 \end{cases}$$

$$\pi = U - W_e$$

$$= \int_0^L \frac{1}{2} EI (u'')^2 dx + \int_0^L \frac{k_f u^2}{2} dx + \frac{K_s}{2} u^2(L) + \int_0^L p(x) u dx$$

where k_f is the subgrade reaction spring constant per unit length L and K_s is the spring constant for flexible end support.

∴ First variation of π $\delta\pi = \int_0^L \delta F dx$

$$\delta F = \frac{1}{2} EI (u'')^2 \delta u + \frac{k_f u^2}{2} - p u + \frac{K_s}{2} u(L)^2$$

$$\therefore \delta F = \frac{\partial F}{\partial u} \cdot \delta u + \frac{\partial F}{\partial u'} \cdot \delta u' + \frac{\partial F}{\partial u''} \cdot \delta u''$$

$$= \frac{k_f}{2} \times 2u \times \delta u + \frac{K_s}{2} \times 2u(L) \times \delta u(L) - p \times \delta u + \frac{1}{2} EI \times 2(u'') \times \delta u''$$

$$\therefore \int_0^L \delta F dx = \int_0^L (k_f u - p) \delta u + \int_0^L EI u'' \delta u'' + K_s u(L) \times \delta u(L)$$

Integrate by parts (the second term):

$$\begin{aligned} \int_0^L EI u'' \times \delta u'' &= EI u'' \times \delta u' \Big|_0^L - \int_0^L EI u''' \times \delta u' dx \\ &= EI u'' \times \delta u' \Big|_0^L - EI u''' \cdot \delta u \Big|_0^L \\ &\quad + \int_0^L EI u^{IV} \times \delta u dx \end{aligned}$$

$$\begin{aligned} \therefore \delta\pi = \int_0^L \delta F dx &= \int_0^L (EI u^{IV} + k_f u - p) \delta u + EI u'' \times \delta u' \Big|_0^L \\ &\quad - EI u''' \cdot \delta u \Big|_0^L + K_s u(L) \times \delta u(L) \end{aligned}$$

But $u, u' = 0$ @ $x=0$

$\therefore \delta u, \delta u' = 0$ @ $x=0$

$$\therefore \int_0^L (EI u'''' + k_f u - P) \delta u \, dx + EI u'' \times \delta u' \Big|_{x=L} - EI u'' \cdot \delta u \Big|_{x=L} + K_s u \cdot \delta u \Big|_{x=L} = 0$$

$\therefore EI u'''' + K_f u - P = 0 \longrightarrow$ governing differential equation.

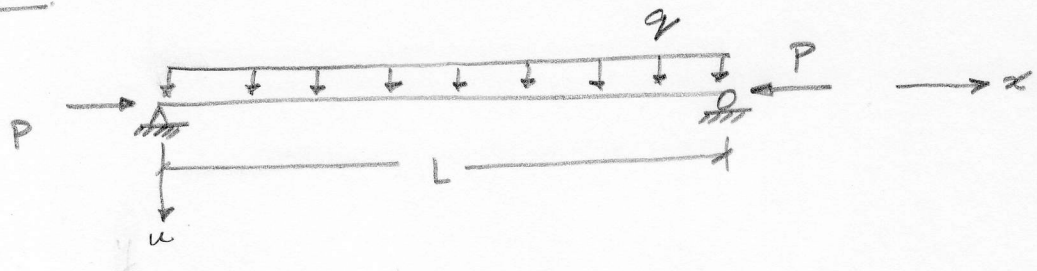
$$\left. \begin{aligned} EI u'' \times \delta u' \Big|_{x=L} &= 0 \\ (K_s u - EI u''') \delta u \Big|_{x=L} &= 0 \end{aligned} \right\} \text{boundary conditions}$$

\therefore At $x=L$

either $EI u'' = 0$
 and either $K_s u - EI u''' = 0$
 natural

or $\delta u' = 0$
 or $\delta u = 0$
 forced

Example:



$$\begin{aligned} \pi &= U - W_e \\ &= \frac{1}{2} EI \int_0^L (u'')^2 dx - \frac{1}{2} P \int_0^L (u')^2 dx - q \int_0^L u dx \\ &= \int_0^L \left(\frac{1}{2} EI (u'')^2 - \frac{1}{2} P (u')^2 - qu \right) dx \end{aligned}$$

$$\delta \pi = \int_0^L \delta F dx$$

$$\begin{aligned} \delta F &= \frac{\partial F}{\partial u} \cdot \delta u + \frac{\partial F}{\partial u'} \cdot \delta u' + \frac{\partial F}{\partial u''} \cdot \delta u'' \\ &= -q \cdot \delta u - P u' \cdot \delta u' + EI u'' \cdot \delta u'' \end{aligned}$$

$$\int_0^L \delta F dx = \int_0^L -q \cdot \delta u - \int_0^L P u' \cdot \delta u' + \int_0^L EI u'' \cdot \delta u''$$

$$-\int_0^L P u' \cdot \delta u' = -P u' \cdot \delta u \Big|_0^L + \int_0^L P u'' \cdot \delta u dx$$

$$\int_0^L EI u'' \cdot \delta u'' = EI u'' \cdot \delta u' \Big|_0^L - EI u''' \cdot \delta u \Big|_0^L + \int_0^L EI u^{IV} \cdot \delta u dx$$

$$\therefore \delta \pi = \int_0^L \delta F \, dx = \int_0^L (EI u^{iv} + Pu'' - q) \delta u \, dx$$

$$+ (-Pu' - EI u''') \delta u \Big|_0^L + EI u'' \cdot \delta u' \Big|_0^L$$

If $\delta \pi = 0 \rightarrow$ Equilibrium.

$$\therefore EI u^{iv} + Pu'' - q = 0 \quad \leftarrow \text{governing differential equation}$$

$Pu' + EI u''' = 0 \quad \text{or} \quad \delta u = 0$	$@ \quad x = 0$ $\quad \quad \quad \& \quad x = L$	Boundary condition 1
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$EI u'' = 0 \quad \text{or} \quad \delta u' = 0$	$@ \quad x = 0$ $\quad \quad \quad \& \quad x = L$	Boundary condition 2
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